



Progress and challenges in the development of the composite wedge localization element



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Abstract



The composite wedge localization element:

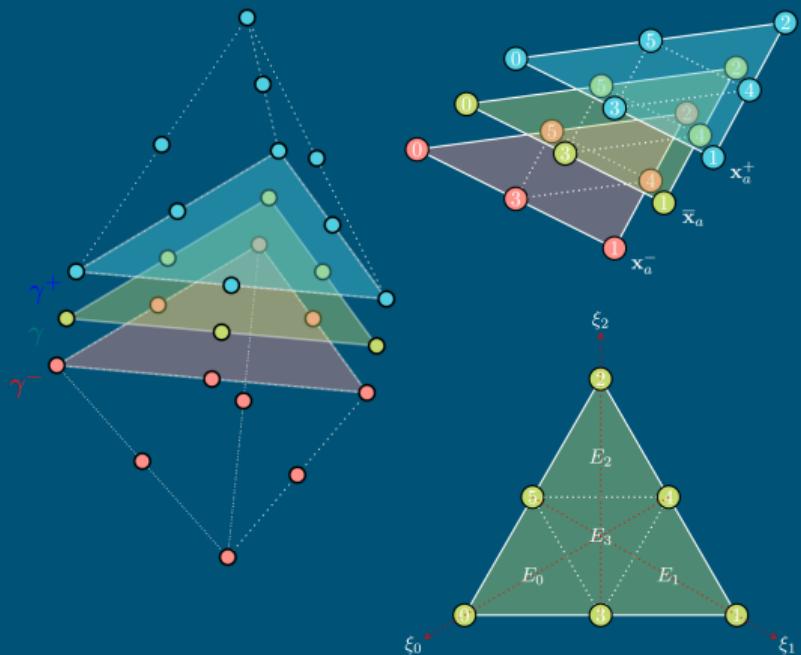
- Composite tetrahedron compatible.
- Regularizes sub-grid localization.
- Extending to fracture and failure.

Progress:

- Lower-order projections.
- Rigid-body modes.

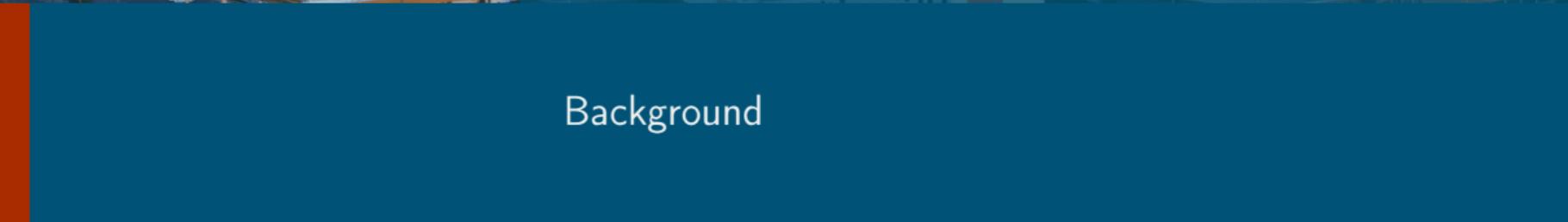
Challenges:

- Pressure field stability.
- Decoupling length scales.
- Implicit solve convergence.





Background



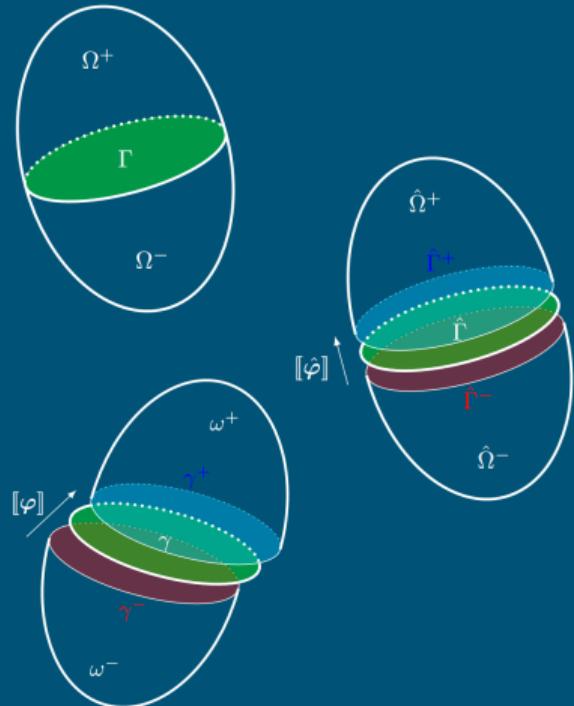
Energy functional



With both solid and localization elements [1–3]:

$$\begin{aligned}\Pi[\varphi, \bar{\mathbf{F}}, \bar{\mathbf{P}}] = & \sum_{\pm} \int_{\Omega^{\pm}} A(\bar{\mathbf{F}}, \mathbf{Z}) dV + \int_{\Gamma} A(\bar{\mathbf{F}}, \mathbf{Z}) h dS \\ & + \sum_{\pm} \int_{\Omega^{\pm}} \bar{\mathbf{P}} : (\mathbf{F} - \bar{\mathbf{F}}) dV + \int_{\Gamma} \bar{\mathbf{P}} : (\mathbf{F} - \bar{\mathbf{F}}) h dS \\ & - \sum_{\pm} \int_{\Omega^{\pm}} \rho_0 \mathbf{B} \cdot \varphi dV - \sum_{\pm} \int_{\partial_T \Omega^{\pm}} \mathbf{T} \cdot \varphi dS\end{aligned}$$

- Lagrange multiplier $\bar{\mathbf{P}}$ enforces $\bar{\mathbf{F}} = \mathbf{F}$, where $\bar{\mathbf{P}} = \mathbf{P} = \partial A / \partial \mathbf{F}$ is also enforced.
- Localization element thickness h required for both integration and normalization.





Let $\mathbf{x}(t) = \varphi(\xi; t)$ and $\mathbf{X} = \mathbf{x}(0) = \varphi_0(\xi)$.

- Deformation from jump [4]:

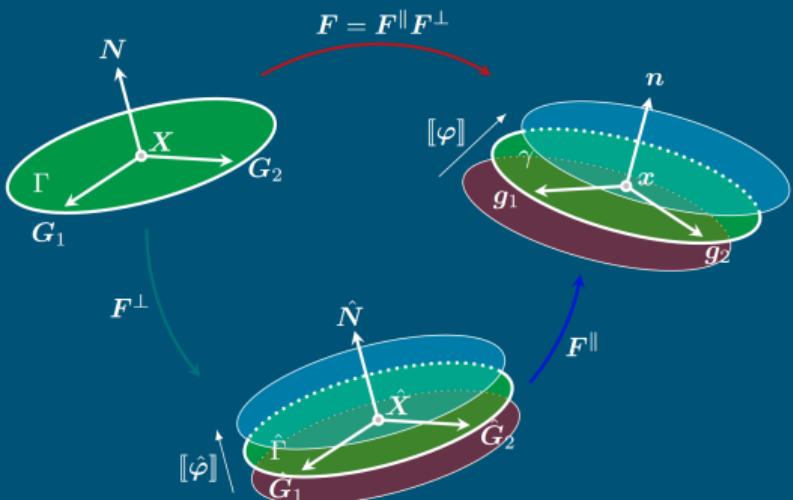
$$\mathbf{F}^\perp = \mathbf{I} + \frac{[\hat{\varphi}]}{h} \otimes \mathbf{N} \quad \text{and} \quad [\varphi] = \mathbf{F}^\parallel [\hat{\varphi}]$$

- Deformation from surface:

$$\mathbf{F}^\parallel = \partial_\mu \varphi \otimes \partial^\mu \varphi_0 + \mathbf{n} \otimes \mathbf{N}$$

- Resulting additive decomposition:

$$\mathbf{F} = \mathbf{F}^\parallel \mathbf{F}^\perp = \mathbf{F}^\parallel + \frac{[\varphi]}{h} \otimes \mathbf{N}$$



Fundamentally different from cohesive surface elements [5].

Element discretization



Let $\tilde{\mathbf{x}}_a = \frac{1}{2}(\mathbf{x}_a^+ + \mathbf{x}_a^-)$ and $[\mathbf{x}_a] = \mathbf{x}_a^+ - \mathbf{x}_a^-$.

- Subtriangles project to linear element:

$$\bar{\mathbf{A}} = \lambda_\alpha \left(\int_{\Gamma_E} \lambda_\alpha \lambda_\beta \mathbf{I} dS \right)^{-1} \int_{\Gamma_E} \lambda_\beta \mathbf{A} dS$$

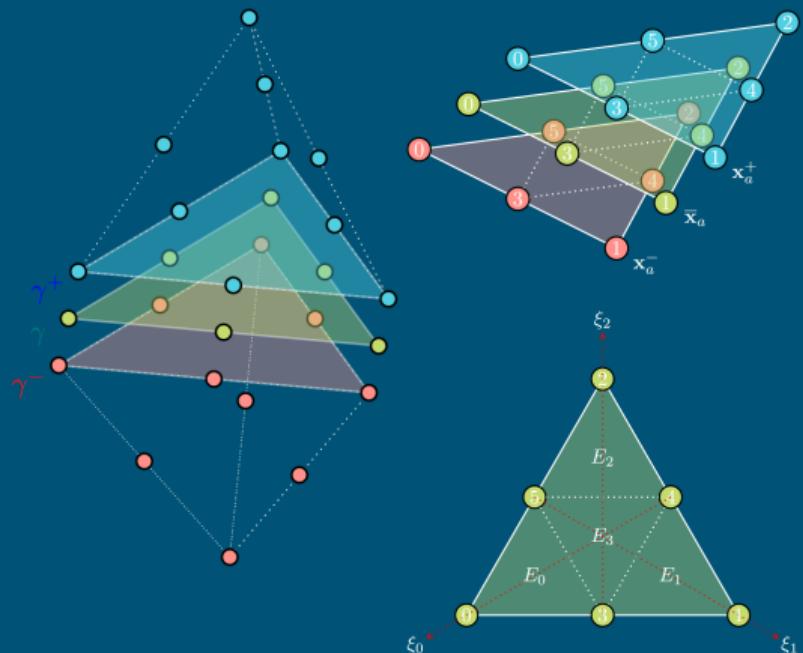
- Projected gradient operators:

$$\bar{\mathbf{F}} = \bar{\mathcal{B}}_a^{\parallel} \tilde{\mathbf{x}}_a + \bar{\mathcal{B}}_a^{\perp} [\mathbf{x}_a]$$

- Nodal forces, quasi-traction-separation:

$$f_a^{\pm} = \frac{1}{2} \int_{\Gamma} \mathbf{P} : \bar{\mathcal{B}}_a^{\parallel} h dS \pm \int_{\Gamma} \mathbf{P} \bar{\mathbf{N}}_a dS$$

Implemented in Sierra/SolidMechanics [6].





Progress



Volumetric locking:

- Observed in nearly incompressible flow.
- Manifested as oscillatory pressure fields.

Mitigation technique [3]:

- Lower-order projection of the Jacobian.

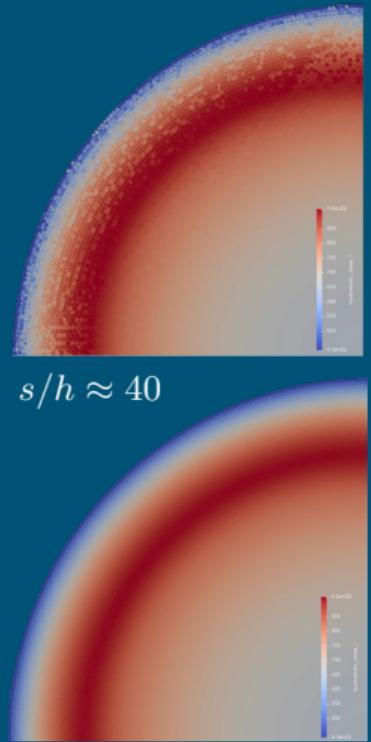
$$\tilde{\mathbf{F}} = \left(\frac{\bar{J}^*}{\bar{J}} \right) \bar{\mathbf{F}} \quad \text{and} \quad \bar{J}^* = \frac{1}{V_\Omega} \int_\Omega \bar{J} \, dV$$

- Lower-order projection of the pressure.

$$\bar{p}^* = \frac{1}{\bar{J}^* V_\Omega} \int_\Omega \frac{\text{tr}(\tilde{\mathbf{P}} \tilde{\mathbf{F}}^T)}{3} \, dV$$

- Corresponding adjusted nodal forces.

$$\mathbf{f}_a = \int_\Omega \left(\tilde{\mathbf{P}} - \frac{1}{3} \text{tr}(\tilde{\mathbf{P}} \tilde{\mathbf{F}}^T) \tilde{\mathbf{F}}^{-T} + \bar{J} \bar{p}^* \tilde{\mathbf{F}}^{-T} \right) \cdot \bar{\mathbf{B}}_a \left(\frac{\bar{J}^*}{\bar{J}} \right)^{1/3} \, dV$$



Rigid-body modes



Surface-separating finite elements:

- Localization elements.
- Cohesive surface elements.

Composite surface-separating finite elements:

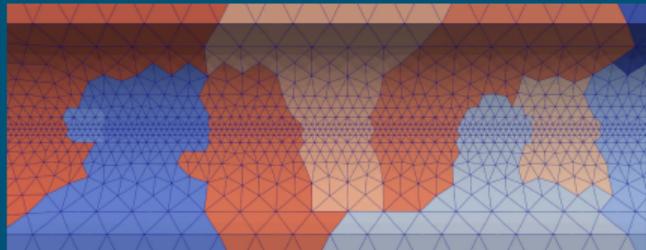
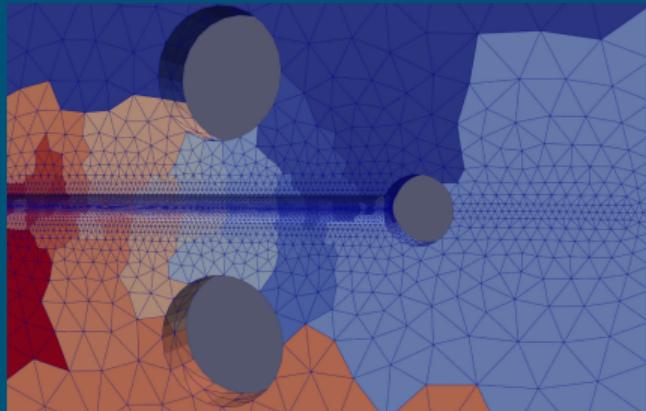
- Additional rigid-body modes.

Using lower-order volumetric projections:

- Additional low-energy modes?
- Stabilization is turned off here.

Parallel decompositions:

- No rigid-body modes in the final assembly as long as the ham stays in the sandwich.
- Need info across processor boundaries.





Challenges



Pressure field stability



Large ratios of s/h disrupts pressure fields:

- Oscillatory or downright nasty.
- Visible effects after significant plasticity.
- Refinement typically alleviates the issue.

An issue for any localization element, so far:

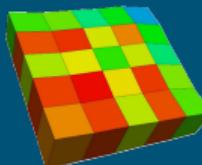
- Hexahedral localization element.
- Composite wedge localization element.

Is there always a point of instability?

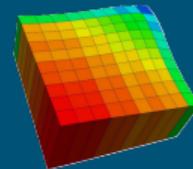


$s/h \approx 4$

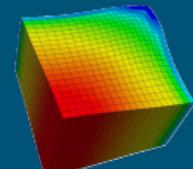
$s/h = 24$



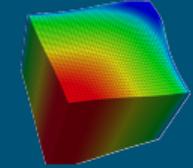
$s/h = 12$



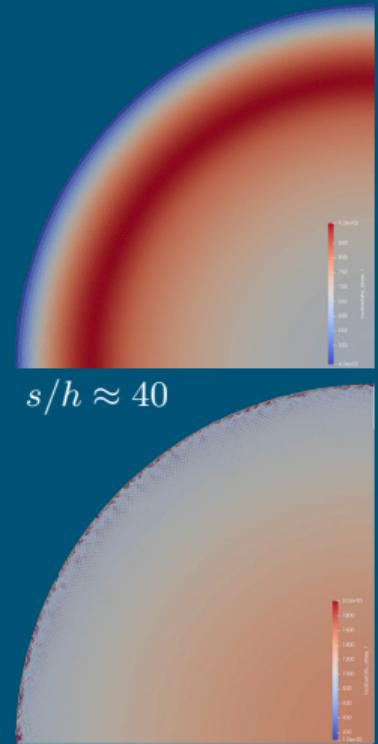
$s/h = 6$



$s/h = 3$



$s/h \approx 40$



Decoupling length scales



Weight contributions separately [7]:

$$f_a^\pm = \frac{1}{2} \int_{\Gamma} \mathbf{P} : \bar{\mathbf{B}}_a^{\parallel} t \, dS \pm \int_{\Gamma} \mathbf{P} \bar{\mathbf{N}}_a \, dS$$

Or try to explicitly retain variational structure:

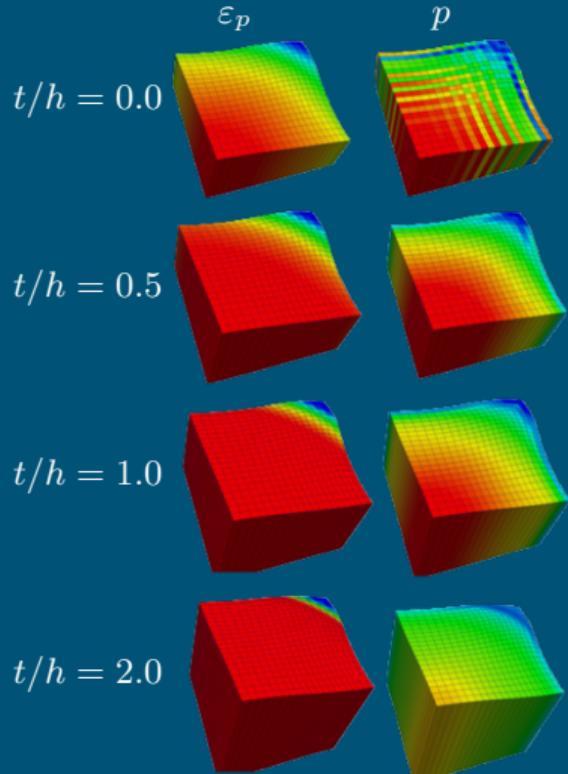
$$\int_{\Gamma} A(\mathbf{F}^{\parallel}, \mathbf{Z}) t \, dS + \int_{\Gamma} [A(\mathbf{F}, \mathbf{Z}) - A(\mathbf{F}^{\parallel}, \mathbf{Z})] h \, dS$$

Surface element, quasi-traction-separation, extra:

$$f_a^\pm = \frac{1}{2} \int_{\Gamma} \mathbf{P}^{\parallel} : \bar{\mathbf{B}}_a^{\parallel} t \, dS \pm \int_{\Gamma} \mathbf{P} \bar{\mathbf{N}}_a \, dS$$

(ignore?) $\pm \int_{\Gamma} (\mathbf{P} - \mathbf{P}^{\parallel}) : \bar{\mathbf{B}}_a^{\parallel} h \, dS$

Is any of this fair in the first place?



Implicit solve convergence



Explicit integration analyses:

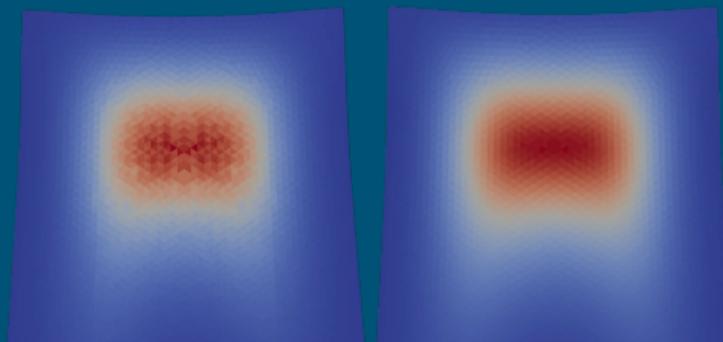
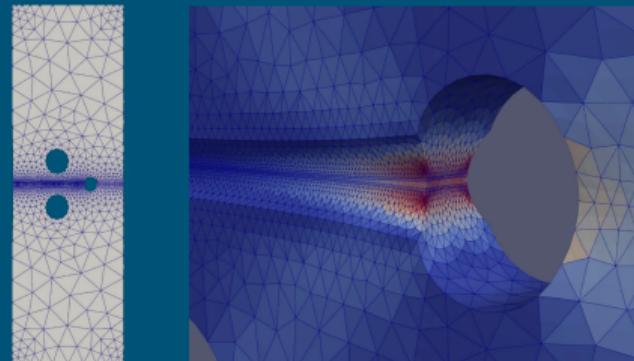
- Complicated by massless elements.
- Less desirable in certain cases.

Implicit integration analyses:

- Sometimes the fields look great, and the damage evolution is "smooth" but it just will not converge!
- Currently a work-in-progress [7-10].

Failure modeling is hard! Who knew?

- Need more refinement?
- Need non-local damage model?
- Something else happening?



Conclusion



The composite wedge localization element:

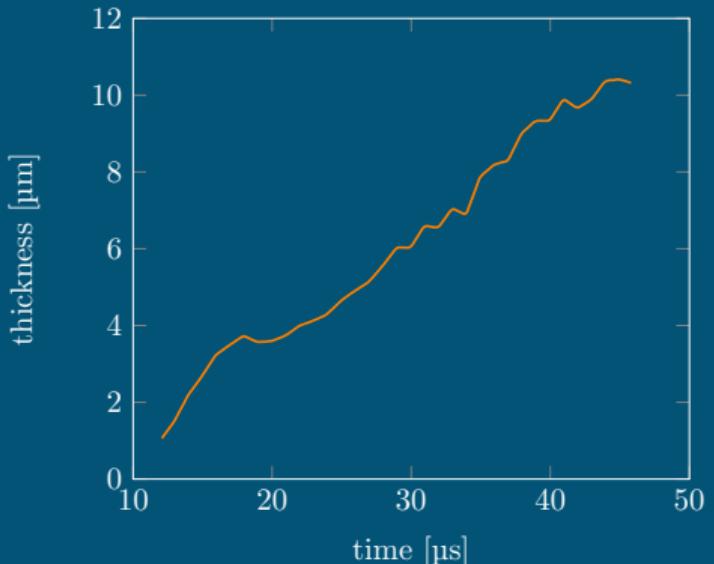
- Composite tetrahedron compatible.
- Original development finished previously.
- Newly implemented lower-order projections.

Ratio of element size to thickness (s/h) issue:

- Is mesh refinement always possible?
- Will scaling the membrane forces work?
 - Which way should they be scaled?
- Does h need to grow as a field [4]?

Implicit integration analyses:

- Is there something preventing convergence?
- Or is this simply a difficult problem to solve?



References



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